## **Amit Medhavi and U. K. Singh**

Department of Mechanical Engineering, Kamla Nehru Institute of Technology, Sultanpur-228118, India.

## **Ajai S. Yadav**

Department of Mathematics, SMS Institute of Technology, Sultanpur Road, Lucknow, India.

**Abstract-**The flow of a particle-fluid suspension in a non-uniform tube induced by sinusoidal peristaltic wave motion of the wall has been investigated. The equations governing the flow for both the fluid and particle phases have been solved and the expressions for the flow rate, pressure rise and friction force has been derived. It is shown that pressure rise increases with the particle concentration but decreases with increasing flow rate. In addition, the pressure rise increases indefinitely with amplitude ratio for any given set of other parameters. Pressure rise assumes considerably smaller magnitude in a non-uniform tube as compared to its corresponding magnitude in uniform diameter tube. Friction force possesses the characteristics opposite to the pressure rise for any given set of parameters.

**Key Words-** Particle concentration, amplitude ratio, flow rate, pressure rise, friction force

## **INTRODUCTION**

In both the physiological and mechanical situations, fluid transport by means of a progressive wave of area contraction or expansion along the wall of a distensible duct containing liquid or mixture, has been the subject of scientific and engineering research for over four decades since the first investigation of Latham (1966). Physiologists term the phenomenon of such transport as peristalsis. Peristaltic pumping has been found to be involved in many biological organs including the vasomotion of small blood vessels such as arterioles, venules and capillaries (Srivastava and Srivastava, 1984), besides its practical applications involving biomechanical systems such as heart-lung machine, finger and roller pumps, etc. Jaffrin and Shapiro (1971) explained the basic principles and clearly brought out the significance of various parameters governing the flow. A summary of most of the experimental and theoretical investigations, reported up to the year 1983, arranged according to the fluid, the Reynolds number, the wave number, the amplitude ratio and the wave shape, was presented in an excellent article by Srivastava and Srivastava (1984). Srivastava and Saxena (1995) have well referenced the important contributions to the topic between the years 1984 and 1994. Some of the recent years studies include the investigations of Srivastava and Srivastava (1997), Mekheimer et al. (1998), Muthu et al. (2001), Srivastava (2002), Misra and Pandey (2002), Hayat et al. (2002,2003,2004), Mekheimer(2003), Misra and Rao (2004), Hayat et al. (2005), Hayat and Ali (2006 a, b), Srivastava (2007), Medhavi and coworkers (2008 a,b), Hayat and Coworkers (2008 a,b), Ali and Hayat (2008),Medhavi and Singh (2009a,b) and a few others.

 The study of particulate suspension is very useful in understanding of a number of diverse physical problems concerned with powder technology, fluidization, and transport of solid particles by liquid and liquid slurries in chemical and nuclear processing, and metalized liquid fuel slurries for rocketry. The sedimentation of particles in a liquid is of interest in many chemical engineering processes. Recently, interest has developed in applying the theory of particle-fluid suspension to physiological flows as it provides improved understanding of the subjects such as diffusion of protein, the rheology of blood, the swimming of microorganism, the particle deposition on respiratory tract, etc. A large number of researches on the subject are referenced in Srivastava and Srivastava (1989).

 Peristaltic pumping of a particle-fluid mixture has been investigated by Hung and Brown (1976), Srivastava and coworker (1989, 1997, 2002), Mekheimer et al. (1998), Medhavi and Singh (2008 b; 2009a,b) and several others. Barring the few (Gupta and Seshadri, 1976; Srivastava and coworkers, 1982, 1983, 1985, 1988; Mekheimer, 2002, etc.), most of the studies in the literature have been conducted in uniform geometry only whereas it is well known that in most of the practical applications, the flow geometry is found to be non-uniform. With increasing interest in particulate suspension flow due to its applications to diverse physical problems, the present study is therefore devoted to study the flow of a particulate suspension in a non-uniform tube induced by sinusoidal peristaltic waves. In view of the theoretical model for blood flow applied in Srivastava and Srivastava (2009) to discuss the effect of hematocrit on flow characteristics in a catheterized artery, it is strongly believed that the study conducted here may be applied to discuss the peristaltic induced flow of blood in small vessels of varying diameter.

### **FORMULATION OF THE PROBLEM**

Consider the flow of a particulate suspension through a diverging tube with a sinusoidal wave travelling down its wall. The geometry of the wall surface is described (Fig.1) as

H (z, t) = a (z) + b sin 
$$
\frac{2\pi}{\lambda}
$$
 (z-ct), (1)

with 
$$
a(z) = a_0 + kz,
$$
 (2)

where  $a(z)$  is the tube radius at any axial distance z from the inlet,  $a_0$  is radius of the tube at the inlet (i.e., at  $z=0$ ),  $k \ll 1$ ) is a constant whose magnitude depends on the length of the tube, exit and inlet dimensions, b is the amplitude of the wave,  $\lambda$  is the wavelength, c is the wave propagation speed and t is the time.



# Fig. 1 Flow geometry of peristaltic waves in a non-uniform tube.

 Using a continuum approach, the equations governing the linear momentum and the mass for both the fluid and particulate phases are expressed (Drew, 1979; Srivastava and coworkers, 1989, 2009) as

$$
(1-C) \rho_f \left\{ \frac{\partial u_f}{\partial t} + u_f \frac{\partial u_f}{\partial z} + v_f \frac{\partial u_f}{\partial r} \right\}
$$
  
= -(1-C)  $\frac{\partial p}{\partial z} + (1-C) \mu_s (C) \nabla^2 u_f + CS (u_p - u_f),$  (3)

$$
(1-C)\rho_f \left\{ \frac{\partial v_f}{\partial t} + u_f \frac{\partial v_f}{\partial z} + v_f \frac{\partial v_f}{\partial r} \right\} =
$$
  
-(1-C)\frac{\partial p}{\partial r} + (1-C)\mu\_s (C)(\nabla^2 - \frac{1}{r^2})v\_f + CS(v\_f - v\_f),  

$$
\frac{1}{r} \frac{\partial}{\partial r} \left[ r (1-C)v_f \right] + \frac{\partial}{\partial z} \left[ (1-C)u_f \right] = 0.
$$
 (5)

$$
C \rho_p \left\{ \frac{\partial u_p}{\partial t} + u_p \frac{\partial u_f}{\partial z} + v_p \frac{\partial u_p}{\partial r} \right\} = -C \frac{\partial p}{\partial z} + CS (u_f - u_p), \tag{6}
$$

$$
C \rho_p \left\{ \frac{\partial v_p}{\partial t} + u_p \frac{\partial v_f}{\partial z} + v_p \frac{\partial v_p}{\partial r} \right\} = -C \frac{\partial p}{\partial r} + CS(v_f - v_p), \tag{7}
$$

$$
\frac{1}{r}\frac{\partial}{\partial r}\left[rC v_p\right] + \frac{\partial}{\partial z}\left[C u_p\right] = 0,\tag{8}
$$

where  $\nabla^2 \equiv (1/r) \partial / \partial r (r \partial / \partial r) + \partial^2 / \partial z^2$  is the two-dimensional Laplacian operator in cylindrical polar coordinate system; (r,z) are cylindrical polar coordinates with z measured along the tube axis and r perpendicular to the axis of the tube;  $(u_f, v_f)$ denotes the fluid phase and  $(u_p, v_p)$  particulate phase velocity components along  $(z, r)$ directions, respectively;  $\rho_f$  and  $\rho_p$  be the actual densities of the material constituting fluid and particulate phases, respectively ; C denotes the volume fraction of the particles, (1-C)  $\rho_f$  be the fluid phase and C  $\rho_p$  the particulate phase densities; p denotes the pressure;  $\mu_s$  (C)  $\approx \mu_s$  be the suspension viscosity and S being the drag coefficient of interaction for the force exerted by one phase on the other. The concentration of the particles is assumed to be small enough so that the particleparticle impact due to the Brownian motion may be neglected. The volume fraction density of the particle, C is chosen to be a constant which is a good approximation for the low concentration of small particles (Batchelor; 1974, 1976).

The empirical relation for the suspension viscosity,  $\mu_s$  and the expression of the drag coefficient of interaction, S for the present study are selected (Charm and Kurland, 1974; Tam, 1969; Srivastava and Srivastava, 2009) as

$$
\mu_{s} = \mu_{0} / (1 - mC),
$$
  
\nm=0.070 exp [2.49C+ (1107/T) exp (-1.69C)],  
\nS = 4.5  $\frac{\mu_{0}}{b_{0}^{2}}$  η(C),  
\n
$$
\eta (C) = \frac{4 + 3(8C - 3C^{2})^{1/2} + 3C}{(2 - 3C)^{2}},
$$
\n(10)

where  $\mu_0$  is the fluid (suspending medium) viscosity,  $b_0$  is the radius of a particle and T is measured in absolute scale of the temperature (K). The viscosity of the suspension expressed by the formula (9) is found to be accurate up to  $C=0.6$  (Charm and Kurland, 1974; Medhavi and Singh, 2008b).

The introduction of the following dimensionless variables

$$
z' = z/\lambda_r t = r/a_0, t' = ct/\lambda, (u'_{f_1} u'_{p}) = (u_{f_1}, u_{p})/c,
$$
  

$$
(v'_{f_1} v'_{p}) = \lambda (v_{f_1}, v_{p})/ca_0, p' = p a_0^2/\lambda c \mu_0, \mu = \mu_s / \mu_0,
$$
  

$$
S' = Sa_0^2 / \mu_0, \delta = a_0 / \lambda, R_e = c \rho_f a_0 / \mu_0,
$$

into equations  $(3)$ – $(8)$ , after dropping the primes, yields

$$
(1-C) \delta R_e \left\{ \frac{\partial u_f}{\partial t} + u_f \frac{\partial u_f}{\partial z} + v_f \frac{\partial u_f}{\partial r} \right\}
$$
  
= - (1-C)  $\frac{\partial p}{\partial z} + (1-C) \mu \left\{ \frac{\partial^2}{\partial z^2} + \frac{1}{r} \frac{\partial}{\partial r} \frac{\delta^2 \partial^2}{\partial z^2} \right\} u_f + CS (u_p - u_f),$  (11)

$$
(1-C)\delta^{3}R_{e}\left\{\frac{\partial v_{f}}{\partial t}+u_{f}\frac{\partial v_{f}}{\partial z}+v_{f}\frac{\partial v_{f}}{\partial r}\right\} =
$$
\n
$$
-(1-C)\frac{\partial p}{\partial r}+(1-C)\mu\delta^{2}\left\{\frac{\partial^{2}}{\partial r}+\frac{1}{r}\frac{\partial}{\partial r}+\delta^{2}\frac{\partial^{2}}{\partial z^{2}}-\frac{1}{r^{2}}\right\}v_{f}+CS\delta^{2}(v_{p}-v_{f})
$$
\n(12)

$$
\frac{1}{r} \frac{\partial}{\partial r} \left[ r \left( 1 - C \right) v_f \right] + \frac{\partial}{\partial z} \left[ (1 - C) u_f \right] = 0, \tag{13}
$$

$$
C \left( \rho_f / \rho_p \right) \delta R_e \left\{ \frac{\partial u_p}{\partial t} + u_p \frac{\partial u_p}{\partial z} + v_f \frac{\partial u_p}{\partial r} \right\} = -C \frac{\partial p}{\partial z} + CS \left( u_f - u_p \right), \tag{14}
$$

$$
C \left( \rho_f / \rho_p \right) \delta^3 R_e \left\{ \frac{\partial v_p}{\partial t} + u_p \frac{\partial v_p}{\partial z} + v_f \frac{\partial v_p}{\partial r} \right\} = -C \frac{\partial p}{\partial r} + CS \delta^2 (v_p - v_f), \quad (15)
$$

$$
\frac{1}{r} \frac{\partial}{\partial r} \Big[ r C v_p \Big] + \frac{\partial}{\partial z} \Big[ C u_p \Big] = 0 \,, \tag{16}
$$

The Reynolds number,  $R_e$  is quite small when the wavelength is large (Jaffrin and Shapiro, 1971), and therefore, the inertial convective acceleration terms may be neglected in comparison to the viscous terms (Shapiro, et al. 1969). Thus, under the long wavelength approximation (i.e.,  $\delta \ll 1$ ), the equations (11)-(16) governing the flow in the laboratory frame of reference in its non-dimensional form reduce to

$$
(1-C)\frac{dp}{dz} = (1-C)\frac{\mu}{r}\frac{\partial}{\partial r}(r\frac{\partial}{\partial r})u_f + CS(u_p - u_f),
$$
\n(17)

$$
C\frac{dp}{dz} = CS(u_f - u_p),
$$
\n(18)

$$
\frac{\mathrm{dp}}{\mathrm{dr}} = 0,\tag{19}
$$

The non-dimensional boundary conditions are

$$
u_f = 0
$$
, at  $r = h = H/a_0 = 1 + \frac{\lambda Kz}{a_0} + \phi \sin 2\pi (z-t)$ , (20)

$$
\frac{\partial u_f}{\partial r} = \frac{\partial u_p}{\partial r} = 0 \quad \text{at} \quad r = 0 \tag{21}
$$

#### **ANALYSIS**

The expressions for the velocity profiles,  $u_f$  and  $u_p$  obtained as the solution of equations (17) and (18), subject to the boundary conditions (20) and (21), are given as

$$
u_{f} = -\frac{1}{4(1-C)\mu} \frac{dp}{dz} \qquad (h^{2} - r^{2}),
$$
 (22)

$$
u_p = -\frac{1}{4(1-C)\mu} \frac{dp}{dz} \quad \left\{ h^2 - r^2 + \frac{4(1-C)\mu}{S} \right\},
$$
 (23)

The instantaneous volumetric flow rate,  $q(z, t)$  is thus calculated as

$$
q (z,t) = 2 \pi \int_{0}^{h} \left[ (1 - C) u_{f} + C u_{p} \right] dr
$$
  
= 
$$
- \frac{\pi}{8(1 - C) \mu} \frac{dp}{dz} \left( h^{4} + \beta h^{2} \right),
$$
 (24)

or

$$
-\frac{dp}{dz} = \frac{8(1-C)\mu)\mu q(z,t)}{\pi h^2 (h^2 + \beta)}
$$
(25)

with  $\beta = 8C(1 - C) \mu / S$ , a non-dimensional suspension parameter.

The pressure rise,  $\Delta p_L(t)$  and the friction force at the wall,  $F_L(t)$  in a tube of length L in their non-dimensional form are given by

$$
\Delta p_{L} (t) = \int_{0}^{L/\lambda} \left(\frac{dp}{dz}\right) dz, \qquad (26)
$$

$$
F_{L}(t) = \int_{0}^{L/\lambda} h^{2} \left( -\frac{dp}{dz} \right) dz, \qquad (27)
$$

An application of equation (25) into equations (26) and (27), yields

$$
\Delta p_{\text{L}}(t) = -8(1 - C)\mu \int_{0}^{L/\lambda} \frac{q(z, t)}{\pi h^{2}(h^{2} + \beta)} dz
$$
 (28)

$$
F_{L}(t) = 8(1-C)\mu \int_{0}^{L/\lambda} \frac{q(z,t)}{\pi(h^{2}+\beta)}dz
$$
 (29)

 The expressions for the pressure rise and the friction force for a particle-fluid mixture in a uniform diameter tube may be obtained from relations (28) and (29) by setting  $K=0$ . In the absence of particle phase (i.e.,  $C=0$ ), the expressions given in equations (28) and (29) reduce to the results obtained in Gupta and Seshadri (1976) for a Newtonian fluid in non-uniform tube . Also, with  $K=0$  and  $C=0$  in equations (28) and (29), the results corresponding to Shapiro et al.(1969) in the laboratory frame of reference are derived from the present study.

## **NUMERICAL RESULTS AND DISCUSSION**

 In order to discuss the results of the study quantitatively, computer codes are now developed for the numerical evaluations of the analytical results for pressure rise,  $\Delta p_1$  (t) and friction force F<sub>1</sub> (t), obtained in equations (28) and (29) respectively, for various parameter values at the temperature of  $25.5^{\circ}$ C. The form of the instantaneous flow rate q (z, t) assumed to be periodic in (z-t) as (Gupta and Seshadri, 1976; Srivastava and Srivastava, 1988; Mekheimer, 2002)

$$
\frac{q(z,t)}{\pi} = Q/\pi - \frac{\phi^2}{2} + \frac{2\lambda k z}{a_0} \phi \sin 2\pi (z-t) + 2 \phi \sin 2\pi (z-t) + \phi^2 \sin^2 2\pi (z-t),
$$
\n(30)

where  $O/\pi$  is the time average of the flow over one period of the wave. The above form of q  $(z, t)$  has been assumed in view of the fact that a constant value of q  $(z, t)$ gives  $\Delta p_{\perp}$  (t) negative and consequently, there would be no pumping action in the tube wall. We now compute the dimensionless pressure rise,  $\Delta p_l$  (t) and friction force,  $F<sub>1</sub>$  (t) over the tube length for various values of the dimensionless flow average,  $Q/\pi$ , amplitude ratio,  $\phi$  and the particle concentration, C, using the form of q (z, t) given in equation (30). The average rise in pressure,  $\Delta p_\parallel$  and friction force, F<sub>L</sub> are then evaluated by averaging  $\Delta p_l$  (t) and  $F_l$  (t), respectively, over one period of the wave. Now using the parameter values (Srivastava and Srivastava, 1984, 1988; Mekheimer, 2002)

$$
a_0 = 0.01
$$
 cm,  $L = \lambda = 10$  cm,  $k = 0.5$   $a_0/L = 0.0005$ ,

the integrals involved in equations (28) and (29) are evaluated numerically and some of the critical results obtained are displayed graphically in Figs. 2-11.

The pressure rise,  $\Delta p_{\parallel}$  (t) decreases with increasing flow rate, Q for a given particle concentration, C but increases with the particle concentration, C for any given flow rate, Q (Fig. 2). The flow characteristic,  $\Delta p_i$  (t) assumes much smaller magnitude in a non-uniform tube than its corresponding magnitude in a uniform tube (Figs. 2 and 3).



average pressure rise,  $\Delta p_i$  (t) versus time averaged flow rate, Q has been shown in Fig.4 in a non- The The uniform tube which indicates a linear relationship between  $\Delta p_{\perp}$  (t) and Q and thus the maximum flow rate is achieved at zero pressure rise and maximum pressure occurs at zero flow rate. The average pressure rise,  $\Delta p_{\parallel}$  increases indefinitely with increasing





different Q and C in a non-uniform tube.



different Q and C in a uniform tube.



amplitude ratio,  $\phi$  for any given flow rate, Q and the particle concentration, C (Fig. 5). The flow characteristic,  $\Delta p_{\parallel}$  assumes a very high asymptotic value at about  $\phi = 0.6$ in both the uniform and non-uniform tube (Figs. 5 and 6).

However,  $\Delta p_l$  assumes considerable smaller magnitude in a non-uniform tube than its corresponding magnitude in a uniform diameter tube. The average pressure rise,  $\Delta p_1$  versus C have been plotted in Figs. 7 and 8 for non-uniform and uniform tubes, respectively.  $\Delta p_l$  is found to be increasing with the particle concentration, C at zero flow rate for a given amplitude ratio,  $\phi$ . The nature in the variation of  $\Delta p_L$  with C is highly influenced with decreasing amplitude ratio,  $\phi$  for any given non-zero value of the average flow rate, Q. One notices that  $\Delta p_1$  increases with amplitude ratio,  $\phi$  for any given flow rate, Q (Figs.7 and 8). $\Diamond$ 



Fig. 8 Variation of pressure rise, 

<sub>L</sub> with particle concentration, C for different Q and  $\phi$  in a uniform tube.



The friction force,  $F<sub>1</sub>$  (t) increases with the time averaged flow rate, Q for a given particle concentration, C, but decreases with increasing particle concentration, C for any given value of the flow rate, Q (Fig. 9). Friction force,  $F_1$  (t) assumes higher magnitude in uniform tube than its



corresponding value in non-uniform tube (Fig. 9 and 10). It is noticed that the average friction force  $F<sub>L</sub>$  increases with the flow rate, Q for any given particle concentration, C and the amplitude ratio,  $\phi$ . A linear relationship between the average friction force, FL and the flow rate, Q is noticed from Fig.11. An inspection of Figs. 2 and 9 reveals that the friction force,  $F<sub>1</sub>$  (t) possesses character opposite to that of the pressure rise,  $\Delta p_i$  (t) for any given set of parameters. A similar conclusion is drawn in the case of their averaged values of pressure rise,  $\Delta p_l$  and friction force,  $F_L$  from Figs.4 and 11.

#### **CONCLUSIONS**

 The flow of a particle-fluid mixture by means of peristaltic wave motion of the wall in a non-uniform tube has been discussed. The pressure rise decreases with the increasing flow rate for a given particle concentration but increases with the particle concentration for any given flow rate. A linear relationship exists between averaged pressure rise and the averaged flow rate. The friction force possesses the characteristics opposite to that of the pressure rise with respect to any parameter. The flow characteristics (pressure rise, averaged pressure rise, friction force, and averaged friction force) assume much smaller magnitude in a non-uniform tube than its corresponding value in uniform diameter tube. The flow equations considered to conduct the study corresponds to the macroscopic two-phase blood flow in narrow arteries. From the published literature, it is known that peristalsis contributes significantly towards the flow in small vessels. It is therefore strongly believed that the results of the present analysis may be applied to explain the peristaltic pumping of blood in small arteries.

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## **NOMENCLATURE**